## MECE 3321: Mechanics of Solids Chapter 12

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# Slope & Displacement by Integration

From Calculus, the radius of curvature is defined where v=f(x)

$$\frac{1}{\rho} = \frac{\frac{d^2 v}{dx^2}}{\left[1 + \left(\frac{dv}{dx}\right)^2\right]^{3/2}}$$

In many engineering applications,  $\frac{dv}{dx} \ll 1$ 

$$\frac{1}{\rho} = \frac{d^2v}{dx^2}$$

Apply relation between radius of curvature and flexure formula

$$\frac{M}{EI} = \frac{d^2v}{dx^2}$$

# Slope & Displacement by Integration

Equation (1)

 $M(x) = EI \frac{d^2 v}{dx^2}$ 

Integrating the bending moment diagram twice will give you the deflection at any point along the beam.

Recall the relationship between a distributed load, shear force, and bending moment:

$$\frac{dM}{dx} = V \qquad \qquad \frac{dV}{dx} = -w(x)$$

Thus,

$$V(x) = EI \frac{d^3 v}{dx^3}$$
$$-w(x) = EI \frac{d^4 v}{dx^4}$$

Note: Equations 1, 2, & 3 are only valid when El is constant along the beam.

Equation (2) Equation (3)





Recall, a positive sign convention for beam bending will cause the beam to "hold water"  $$_{\pm w}$$ 







# Determine the equations of the elastic curve using the $x_1$ and $x_2$ coordinates. El is constant.

## Method of Superposition

Superposition is used to determine the slope or deflection at certain points of a beam due to several loads whose effect is first separately computed and then added to find total values.

| Beam  | Slope  | Deflection   | Elastic Curve   |
|---|--|--|---|
| $\begin{array}{c} \begin{array}{c} & \\ & \\ & \\ \end{array} \end{array} \xrightarrow{P} \begin{array}{c} & \\ & \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ & \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ \end{array} \xrightarrow{P} \begin{array}{c} \\ \\ \end{array} \xrightarrow{P} \begin{array}{c} \end{array} \xrightarrow{P} \end{array} \xrightarrow{P} \begin{array}{c} \end{array} \xrightarrow{P} \end{array} \xrightarrow{P} \begin{array}{c} \end{array} \xrightarrow{P} \end{array} P$ | $\theta_{\max} = \frac{-PL^2}{16EI}$                                   | $v_{\max} = \frac{-PL^3}{48EI}$                        | $v = \frac{-Px}{48EI} (3L^2 - 4x^2)$ $0 \le x \le L/2$    |
| $\theta_1$ $\theta_2$ $\alpha_2$ $\alpha_1$ $\alpha_2$ $\alpha_2$ $\alpha_3$  | $\theta_1 = \frac{-Pab(L+b)}{6EIL}$ $\theta_2 = \frac{Pab(L+a)}{6EIL}$ | $v \bigg _{x=a} = \frac{-Pba}{6EIL} (L^2 - b^2 - a^2)$ | $v = \frac{-Pbx}{6ELL} (L^2 - b^2 - x^2)$ $0 \le x \le a$ |









### Problem 12-121

Determine the deflection at the end B of the clamped A-36 steel strip. The spring has a stiffness of k = 2 N/mm. The strip is 5 mm wide and 10 mm high. Also, draw the shear and moment diagrams for the strip.



