

1.) zero equation \Rightarrow various eddy viscosity models can be tried

$$\mu_t = \text{func}(\gamma, \bar{u}, \frac{\partial \bar{u}}{\partial y}, \text{etc.})$$

2.) one equation

add turbulent kinetic energy equation

$$\overset{\text{energy}}{\text{mass}} \rightarrow K = \frac{1}{2} (\overline{u'u'} + \overline{v'v'} + \overline{w'w'})$$

boundary layer form of equation presented earlier

$$\bar{u} \frac{\partial K}{\partial x} + \bar{v} \frac{\partial K}{\partial y} = - \frac{\partial}{\partial y} \left[\overline{v' \left(\frac{1}{2} u_i' u_i' + P'/\rho \right)} \right] + \frac{\gamma \mu_t}{\rho} \frac{\partial \bar{u}}{\partial y} - \epsilon \quad E_2 (6-26)$$

$$\epsilon = \nu \overline{\frac{\partial u_i' \partial u_j'}{\partial x_j \partial x_j}} \quad \text{turbulent dissipation} \quad \left(\frac{\text{power}}{\text{mass}} \right)$$

let L_e = turbulent length scale
(effective eddy size)

char. eddy velocity $= K^{1/2}$

suggests then

$$\epsilon \sim \frac{\text{drag} \times \text{velocity}}{\text{mass}} \sim \frac{\tau^2 (\rho K^{3/2}) \times K^{1/2}}{\rho L^3} \sim \frac{K^{3/2}}{L}$$

$$\sim \frac{\partial K}{\partial y}$$

$$\tau = (\mu + \mu_t) \frac{\partial \bar{u}}{\partial y}$$

further modeling for $\mu_t = \text{func}(K, \dots)$
and $L_e = \text{func}(K, \dots)$

for math closure

then have three P.D.E.'s for \bar{u}, \bar{v}, κ

author points out that one-equation model no longer popular

3. two equations κ - ϵ

author indicates that this is currently most widely used approach (as of ~1990)

non boundary layer form in text, use log-law to match near the wall

two-dimensional B.L. form

Eqs 6-109a and 6-109b

$$\bar{u} \frac{\partial \kappa}{\partial x} + \bar{v} \frac{\partial \kappa}{\partial y} \approx \frac{\partial}{\partial y} \left[\left(\nu + \frac{\nu_t}{\sigma_\kappa} \right) \frac{\partial \kappa}{\partial y} \right] + \nu_t \left(\frac{\partial \kappa}{\partial y} \right)^2 - \epsilon - 2\nu \left(\frac{\partial \kappa^{1/2}}{\partial y} \right)^2 \quad (6-109a)$$

$$\bar{u} \frac{\partial \epsilon}{\partial x} + \bar{v} \frac{\partial \epsilon}{\partial y} \approx \frac{\partial}{\partial y} \left[\left(\nu + \frac{\nu_t}{\sigma_\epsilon} \right) \frac{\partial \epsilon}{\partial y} \right] + C_1 \frac{\epsilon \nu_t}{\kappa} \left(\frac{\partial \bar{u}}{\partial y} \right)^2 - C_2 \frac{\epsilon^2}{\kappa} + 2\nu \nu_t \left(\frac{\partial^2 \bar{u}}{\partial y^2} \right)^2 \quad (6-109b)$$

where, for example

$$\nu_t = C_m \kappa^2 / \epsilon$$

$$C_m = 0.09 \exp \left[- \frac{2.5}{1 + R_T / 150} \right]$$

$$R_T = \kappa^2 / \epsilon \nu \quad \text{"turbulence Reynolds number"}$$

then have four simultaneous P.D.E.'s for

$$\bar{u}, \bar{v}, \kappa, \epsilon$$

4. Reynolds stress equation

next higher order analysis would be to eliminate

$\nu_+ \approx \frac{C_u \kappa^2}{\epsilon}$ and solve for Reynolds stresses directly

e.g. Eq (6-11)

$$\frac{D}{Dt} (\overline{u_i' u_j'}) = \underbrace{D_{ij}}_{\text{diffusion}} + \underbrace{P_{ij}}_{\text{production}} + \underbrace{\overline{\Pi_{ij}}}_{\text{pressure strain}} - \underbrace{\epsilon_{ij}}_{\text{dissipation}} + \nu \nabla^2 (\overline{u_i' u_j'})$$

terms or might require further modeling as a function of $\kappa, \epsilon, \overline{u_i' u_j'}$, etc.

would then have ^{eleven} ~~ten~~ equations for

$$\overline{w}, \overline{u}, \overline{v}, \kappa, \epsilon, \overline{u_i' u_j'} \quad (3D)$$

since eqs. for $\overline{u}, \overline{v}, \kappa, \epsilon, \overline{u_i' u_j'}$ (2.0)

knowing $\overline{u_i' u_j'}$ means eddy viscosity model no longer needed

expect more advanced/sophisticated than κ - ϵ approach

simplification would be to use empirical algebraic equations for $\overline{u_i' u_j'} = \text{func}(\kappa, \epsilon, \text{etc.})$

Eq (6-116)

then back to four P.D.E.'s for $\overline{u}, \overline{v}, \kappa, \epsilon$ ala κ - ϵ approach

beyond this have

5. large eddy simulation

6. direct (model free) numerical simulation

solution of differential equations ordinarily must be done numerically using finite difference techniques, commercial codes are available, may require large memory and CPU time

as an alternative, simpler integral methods, may suffice for many engineering applications

consider steady, two-dim flow over nonporous surface

$$dU_e/dx \neq 0$$

cons. of mass $\left(\frac{\partial \bar{u}}{\partial x} + \frac{\partial \bar{v}}{\partial y} = 0 \right) (U_e - \bar{u})$ apparent!

cons. of x-mom. $\bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} = U_e \frac{dU_e}{dx} + \frac{1}{\rho} \frac{\partial \tau}{\partial y}$

multiply continuity by $(U_e - \bar{u})$ and subtract from momentum

$$-U_e \frac{\partial \bar{u}}{\partial x} + \bar{u} \frac{\partial \bar{u}}{\partial x} - U_e \frac{\partial \bar{u}}{\partial y} + \bar{u} \frac{\partial \bar{u}}{\partial y} + \bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y}$$

$$- \bar{u} \frac{\partial U_e}{\partial x} + \bar{u} \frac{\partial U_e}{\partial x} = U_e \frac{dU_e}{dx} + \frac{1}{\rho} \frac{\partial \tau}{\partial y}$$

$$\cancel{- \bar{u} \frac{\partial U_e}{\partial x}} \quad \uparrow \text{add/subt.} \quad \text{negroup}$$

$$-\frac{1}{\rho} \frac{\partial \tau}{\partial y} = \frac{\partial}{\partial x} (\bar{u} U_e - \bar{u}^2) + (U_e - \bar{u}) \frac{\partial U_e}{\partial x} + \frac{\partial}{\partial y} (U_e \bar{v} - \bar{u} \bar{v})$$

now integrate $\int_0^{\delta} \dots dy$

@ $y=0$ $\tau = \tau_w$, $\bar{u} = 0$, $\bar{v} = 0$

@ $y=\delta$ $\tau \rightarrow 0$, $\bar{u} \Rightarrow U_e$

$$+ \frac{1}{\rho} \tau_w = \frac{\partial}{\partial x} \left[U_e^2 \int_0^{\delta} \left(1 - \frac{\bar{u}}{U_e}\right) \frac{\bar{u}}{U_e} dy \right]$$

$$+ U_e \frac{dU_e}{dx} \int_0^{\delta} \left(1 - \frac{\bar{u}}{U_e}\right) dy + \left(\bar{v} U_e - \bar{u} \bar{v} \right) \Big|_0^{\delta}$$

$$\frac{1}{\rho} \tau_w = \frac{d}{dx} [U_e^2 \theta] + U_e \delta^* \frac{dU_e}{dx}$$

$$\frac{\tau_w}{\rho U_e^2} = \frac{\theta}{U_e} \frac{dU_e}{dx} + \frac{U_e^2}{U_e^2} \frac{d\theta}{dx} + \frac{U_e \delta^*}{U_e^2} \frac{dU_e}{dx}$$

$$\text{or } \boxed{\textcircled{1} \left[\frac{C_f}{2} = \frac{d\theta}{dx} + (2+H) \frac{\theta}{U_e} \frac{dU_e}{dx} \right]} \quad E_2 (6-2B)$$

van Karman integral equation for time-ave. turbulent flow (see pgs: 265-266, for identical laminar result)

for known $U_e(x)$, need two more equations to relate

$$C_f, \theta, H$$

from Coles wall-wake law

$$u^+ = \frac{1}{\kappa} \ln y^+ + B + \frac{2\pi}{\kappa} f(\eta)$$

find relationships between C_f, H, θ, τ

$$\left(\text{integrate for } \delta^*, \theta \right) \quad H = \frac{\delta^*}{\theta} = f_{\text{func}}(C_f, \tau)$$

$$\theta = f_{\text{func}}(C_f, \tau)$$

$$\text{Eq (6-119)} \quad \sqrt{\frac{\tau_w}{\rho}}$$

$$\textcircled{2} \quad \lambda = \frac{U_e}{\sqrt{\tau_w}} = \left(\frac{2}{c_f}\right)^{1/2} = \frac{2 + 3.179\pi + 1.5\pi^2}{\pi(1+\pi)} \frac{H}{H-1}$$

$$\text{also } Re_\theta = \frac{1+\pi}{\pi H} \exp(\pi\lambda - \pi\beta - 2\pi)$$

if eliminate π , they can create curve fits among c_f, Re_θ, H

$$\text{Eq (6-117)} \quad c_f \cong 0.246 (Re_\theta)^{-0.268} \quad 10^{-0.678 H}$$

$$\text{Eq (6-118)} \quad c_f \cong 0.058 Re_\theta^{-0.268} (0.93 - 1.95 \log H)^{1.70}$$

note typo!

$$\textcircled{3} \quad \text{Eq (6-120)} \quad c_f \cong \frac{0.3 \exp(-1.32 H)}{(\log Re_\theta)^{1.74 + 0.31 H}}$$

still one equation short (θ, H, c_f, π)

from correlation of data

$$\textcircled{4} \quad \beta = -0.4 + 0.76\pi + 0.42\pi^2 \quad \text{Eq (6-121)}$$

Clausen's parameter $\beta = \frac{\delta^*}{\tau_w} \frac{dPe}{dx} = - \frac{\delta^*}{\tau_w} \underbrace{U_e}_{\propto U^{*2}} \frac{dU_e}{dx} \times \frac{U_e}{U_e}$

$$\beta = - \frac{\lambda^2 H \theta}{U_e} \frac{dU_e}{dx} \quad \text{Eq (6-122)}$$

note: 1 to 1 correlation
for equilibrium turbulent flows

basically have four equations for four unknowns

one ord. diff. equation and three algebraic for

$$\vartheta, H, C_s, \pi$$

begin with known $\vartheta(0)$

then numerically "march" downstream for any $U_e(x)$ to determine

$$\vartheta(x), C_s(x), H(x), \pi(x)$$

$$\Rightarrow \delta^*(x), \tau_w(x), \bar{u}(x, y), \delta(x)$$

(somewhat analogous to Thwaites method for laminar boundary layer)

could use Runge Kutta, I used less sophisticated predictor-corrector

$$\frac{d\vartheta}{dx} = \frac{1}{\lambda^2} - (2+H) \frac{\vartheta}{U_e} \frac{dU_e}{dx} \quad \text{Eq. (6-28)}$$

$$\vartheta_{m+1}^p = \vartheta_m^c + \left. \frac{d\vartheta}{dx} \right|_m \Delta x$$

$$\vartheta_{m+1}^c = \vartheta_m^c + \left. \frac{d\vartheta}{dx} \right|_{m+1/2} \Delta x$$

$$\text{where } \vartheta_{m+1/2} = (\vartheta_{m+1}^p + \vartheta_m^c) / 2$$

etc.

also

$$H = \frac{1}{1 - a/\lambda} \quad \text{where } a = \frac{2 + 3.179\pi + 1.5\pi^2}{1\pi(1+\pi)}$$

$$C_f = \frac{0.3 \exp(-1.33H)}{\log(Re_a)^{1.74 + 0.31H}}$$

$$\pi = -(0.905) + \sqrt{(0.905)^2 + (0.952 + 2.381\beta)}$$

$$\text{where } \beta = -\lambda^2 H \frac{\partial}{\partial U_e} \frac{dU_e}{dx}$$

$$\lambda = \left(\frac{2}{C_f}\right)^{1/2}$$

→ iterate for given θ

handout for KRMNTBL.FOR $\frac{U_e(x) \rightarrow}{\frac{dx}{L} \rightarrow}$
 on fileserver H:\ME812

note: three algebraic (nonlinear) equations
 relating $\theta/L, C_f, H, \pi$

only one independent

to find C_f for given (θ/L) (for example)

set $M = 0$ and guess (θ/L) until obtain
 proper C_f

questions?

other integral method approaches have been tried

Sec 6-8.1.2 entrainment integral methods
use van Karman integral equation

$$\textcircled{1} \quad \left| \frac{C_g}{2} = \frac{d\theta}{dx} + (2+H) \frac{\theta}{U_e} \frac{dU_e}{dx} \quad E_2(6-28) \right.$$

but now introduce entrainment equation which was derived from continuity earlier

$$\bar{E} = \frac{d\delta}{dx} - \frac{U_e}{U_e} = \frac{1}{U_e} \frac{d}{dx} [U_e (\delta - \delta^*)] \quad E_2(6-31)$$

work of Head (1958), introduced new shape factor

$$H_1 = (\delta - \delta^*) / \theta$$

$$\textcircled{2} \quad \left| \frac{d(U_e \theta H_1)}{dx} = U_e E = U_e F(H_1) \right. \quad \left. \begin{array}{l} \int \\ \text{curved it} \end{array} \right.$$

$$\text{where } F(H_1) = 0.0306 (H_1 - 3.0)^{-0.6169} \\ E_2(6-123)$$

$$\textcircled{3} \quad \left| \text{then relate } H_1 = \text{func}(H) \quad E_2(6-124) \right.$$

$$\textcircled{4} \quad \left| \text{with } C_g = \text{func}(\theta, H) \quad E_2(6-120) \right.$$

four equs. for C_g , H , θ , H_1
 $\int \frac{d\theta}{dx}$ $\int \frac{dH_1}{dx}$

two parameter system, good except for near-separating flows

note: does not use empirical $\beta = \text{func}(\pi)$,

variation of this is approach of Fenzinger, better in general, but not as good for favorable flows

Sec 6-8.2 inner variable integral method
Das (1988)

same as inner variable analysis for flow over flat plate but add wake

$$\frac{\bar{u}}{U^*} = \frac{1}{\kappa} \ln \eta^+ + B + \frac{2\pi}{\kappa} (3\eta^2 - 2\eta^3)$$

$$\eta = y/\delta$$

from continuity $\frac{\partial \bar{u}}{\partial x} + \frac{\partial \bar{v}}{\partial y} = 0$

$$\bar{v} = - \int_0^y \frac{\partial \bar{u}}{\partial x} dy'$$

use to eliminate \bar{u} and \bar{v} in x -mom equation, integrate from $y^+ = 0$ to $y^+ = \delta^+$
so $\tau = \tau_w \Rightarrow 0$

results in ordinary differential equation in x -direction

in non-dim form $\xi = \frac{U^*}{U_e} = \frac{1}{\lambda} = \left(\frac{C_f}{2}\right)^{1/2}$

$$V = U_e/U_0$$

$$x^* = x/L, \quad Re_L = \frac{U_0 L}{\nu}$$

Eq (6-134)

$$\textcircled{1} \quad \frac{d\delta}{dx^*} = -\frac{\rho}{V} \frac{dV}{dx^*} + \frac{1}{V} \frac{T_1 - T_2 + R_L T_3 (T_4 - R_L T_5)}{R_L T_3 (T_6 + T_7) + T_8 + T_9}$$

 $T_1, T_2, \dots, T_9 \Rightarrow$ Table 6-2

$$= \text{func}(V, \frac{dV}{dx^*}, \frac{d^2V}{dx^{*2}}, \delta^+, \pi)$$

need relations for δ^+, π evaluate Coles wall-law law @ $y^+ = \delta^+$

$$\frac{1}{\kappa} \ln \delta^+ + B + \frac{2\pi}{\kappa} = \frac{1}{\xi} \quad \textcircled{1}$$

other is correlation between B and π

$$\rightarrow B = -0.4 + 0.76\pi + 0.42\pi^2$$

where

$$B = \frac{\delta^*}{\tau_w} \frac{dP_e}{dx} = -\frac{\delta^*}{\tau_w} \rho U_e \frac{dU_e}{dx} \quad \text{Eq. 6-98}$$

$$= -\frac{\delta^*}{\frac{\delta^*}{\xi^2} U_e} \frac{dU_e}{dx^*} \left(\frac{\delta^*}{\delta} \right) \times \frac{(1+\pi) U_e^*}{\kappa U_e^*} \times \frac{\nu}{\delta} \times \frac{U_e}{U_e^*}$$

$$B = -\frac{\delta^+}{\xi^2} \frac{(1+\pi)}{\kappa Re_L} \frac{1}{V^2} \frac{dV}{dx^*} \quad \text{Eq. (6-135)} \times \frac{U_0}{U_0} \times \frac{\kappa}{L}$$

← error in text??

three equations (one diff.) to match along

in ξ, π, δ^+

I wasn't successful in getting this method to work, looked up original paper, might be some typos in Table 6-2

note: doesn't use $C_f = f_{unc}(Q, M)$
correlations

integral methods work best for equilibrium or near equilibrium flows

for rapidly changing freestream,

$$\beta = \beta_1 \text{ for } x < x_0$$

$$\beta = \beta_2 \text{ for } x > x_0$$

use "lag" equation where

$$\frac{dQ}{dx} = \frac{C}{\delta} (Q_{\infty} - Q)$$

← asymptotic value

$$C = 0.025$$

Sec 6-8.3 finite difference methods

solve partial differential equations

Sec 6-8.3.1 zero equations

continuity, x-momentum for \bar{u}, \bar{v}

add eddy viscosity $\mu_t = f_{unc}(x, \frac{\partial \bar{u}}{\partial y}, \text{etc.})$

GENBLFOR \Rightarrow implicit marching method

more sophisticated codes than GENBLFOR

e.g. variable gridding, coordinate transformations, wall functions to patch through sublayer and buffer layer

example Cebbi and Smith (1974)

Sec 6-8.3.2 two equations

$k-\epsilon$ Jones and Launder (1972)

Wilcox (1988) suggests $k-w$ instead of

$k-\epsilon$

Sec 6-8.3.3 Reynolds stress equations

various approaches

Launder (1975)

Stanford Conferences provided experimental test cases for comparison

Tanner-based
Head (1958)
Das (1988)
Ferzinger (1982) } integral methods

Cebbi and Smith (1974) } zero equations

Sec 6-8.4 flat plate

$C_f, \mathcal{D}_{vs.} x \rightarrow$ reasonably good agreement, all methods

for detail, $\bar{u}(y), \bar{v}(y), \tau(y)$ etc. need finite diff. methods

Sec 6-8.4.1 visualization and simulation turbulent flow over a flat plate

hydrogen bubble visualization at

$$y^+ = 2, 4, 11.7, 40.7, 162$$

\uparrow \uparrow \uparrow \uparrow \uparrow
 sublayer buffer log layer

photos reversed??

direct numerical simulation



good agreement with experiment

even at "low" Reynolds number,
10 million mesh points!!

Sec 6-8.5 adverse pressure gradient

$$U_e \approx 56.6 x^{-0.28}$$

Karman-based integral method gave better agreement than more advanced approaches

perhaps not surprising since equilibrium turbulent flow

Sec 6-8.6 nonequilibrium separating flows

Cebbi and Smith provided best fit

inverse method for handling separating flows, assume $f^*(x)$ known, solve for $U_e(x)$, integral methods

Sec 6-8.7 "complex" turbulent flows

flow over steps, recirculation



roughness, suction/blowing; periodic free stream velocity, etc.

"no method universally good"

"no method universally bad"

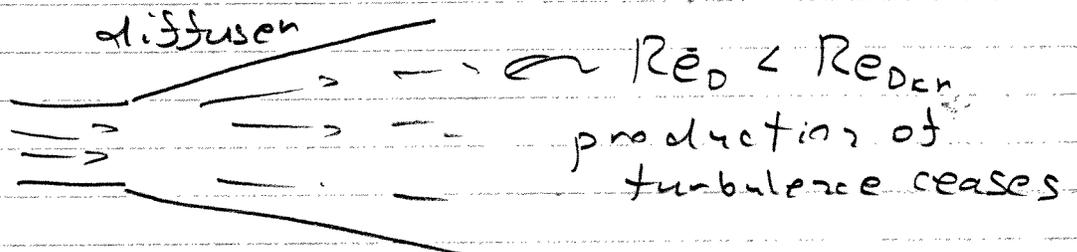
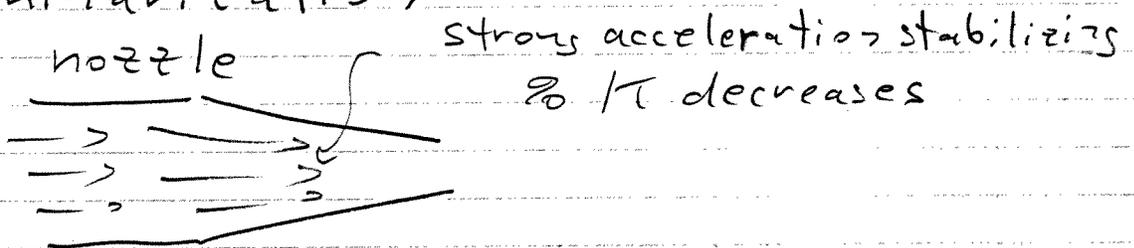
"no correlation between complexity of method and predictive capability"

often simplest integral method the best

might need to use "expert system" approach

obviously, progress continues, much research left to be done

relaminarization



$\int \tau$ -e methods work best for this problem