

VISCOUS FLOW II

Review Material

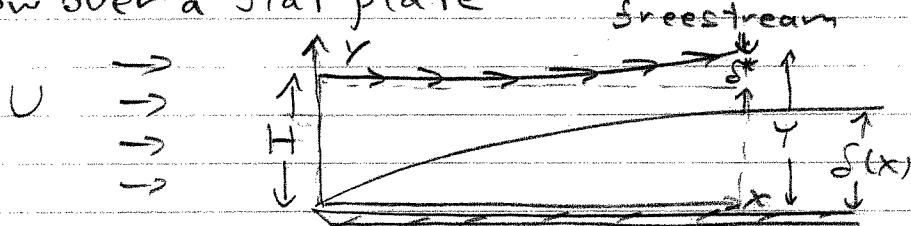
Chap 9 Laminar Boundary Layer Theory

for sufficiently large Reynolds number ($Re_L \geq 1000$), appreciable viscous effects confined to a relatively thin layer near the surface \Rightarrow "viscous boundary layer"

patch together thin viscous boundary layer to inviscid freestream \Rightarrow "boundary layer method"

Sec 9-1 integral analysis of viscous boundary layer

flow over a flat plate



$$\delta(x) = \text{boundary layer thickness}, u|_{y=\delta} = 0.99U$$

$$\delta^*(x) = \text{boundary layer displacement thickness}$$

$$\delta^*(x) = \int_0^{\delta^*} (1 - u/U) dy$$

$$\theta(x) = \text{boundary layer momentum thickness}$$

$$\theta(x) = \int_0^{\delta} (1 - u/U) (u/U) dy$$

$$H = \text{boundary layer shape factor} = \delta^*/\theta$$

for flat plate ($\partial P/\partial x = 0$, $U = \text{const.}$)

$$T_w = \rho U^2 \frac{d\theta}{dx}, C_f = \frac{T_w}{\frac{1}{2} \rho U^2} = 2 \frac{d\theta}{dx}$$

$$C_D = \frac{D'}{\frac{1}{2} \rho U^2 L} = 2 \frac{\theta(L)}{L}$$

is substitute $u \approx U [2(\frac{y}{\delta}) - (\frac{y}{\delta})^2]$ find

$$\frac{\delta}{x} \approx \frac{5.5}{\sqrt{Re_x}}, \dots \text{etc.}$$

in the boundary layer ($Re_x \gg 1$)

$$y = \delta \sim x/\sqrt{Re_x} \Rightarrow y \ll x$$

$$u \sim U, v \sim \frac{U}{\sqrt{Re_x}} \Rightarrow v \ll u$$

$$\frac{\partial u}{\partial x} \sim \frac{U}{x}, \frac{\partial u}{\partial y} \sim \frac{U}{\delta} \sim \frac{U}{x} \sqrt{Re_x} \Rightarrow \frac{\partial u}{\partial x} \ll \frac{\partial u}{\partial y}$$

$$\frac{\partial v}{\partial x} \sim \frac{U}{x} \frac{1}{\sqrt{Re_x}}, \frac{\partial v}{\partial y} \sim \frac{U}{\delta} \sim \frac{U}{\sqrt{Re_x}} \frac{1}{x} \Rightarrow \frac{\partial v}{\partial x} \ll \frac{\partial v}{\partial y}$$

skip Secs. 4-1.6 \rightarrow 4-1.7

Sec 4.2 the laminar boundary layer equations

continuity $\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$

x-mom $\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = - \frac{\partial p}{\partial x} + \mu \frac{\partial^2 u}{\partial y^2}$

y-mom $\frac{\partial p}{\partial y} = 0$

energy $\rho c_p \left(\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) = k \frac{\partial^2 T}{\partial y^2} + \mu \left(\frac{\partial u}{\partial y} \right)^2$

observations

- applicable for curved surfaces if $R \gg \delta$
- flow direction transport negligible ($\frac{\partial^2 u}{\partial x^2} \ll \frac{\partial^2 u}{\partial y^2}$)
- for constant fluid properties, energy equation is decoupled

- $P = \text{func}(x)$ only and is known from freestream conditions
- x -dir mom. equ. is parabolic for $U(x,y)$, numerical marching schemes can be used in $+x$ direction (likewise for $T(x,y)$)
- boundary conditions @ $y=0$ $U=0, V=0, T=T_w$
@ $y \rightarrow \text{large}$ $U=U_\infty, T=T_\infty$
- viscous stresses $\tau_{xy}' \approx \mu \frac{\partial U}{\partial y}$
 $\tau_{xy}' \gg \tau_{xx}, \tau_{yy}'$
- $Re \gtrsim 1000$
- for $Re \gtrsim 10^6 \Rightarrow$ instability, turbulence
- if $\frac{\partial U}{\partial x} < 0, \frac{\partial P}{\partial x} > 0$, possibility of boundary layer separation ($T_w \rightarrow 0$)

Blasius solution \Rightarrow flow over a flat plate

$$U = \text{const}, \frac{\partial P}{\partial x} = 0$$

similarity variable $\eta = y \sqrt{\frac{U}{2 \nu x}}$

similarity function $f(\eta)$

$$U = U f'(\eta), V = \sqrt{\frac{U}{2x}} [\eta f''(\eta) - f(\eta)]$$

Blasius equation $f''' + f f'' = 0$

$$f(0) = 0, f'(0) = 0, f'(\eta \rightarrow \text{large}) = 1$$

displaced B.C.s, solve by trial and error
using Runge-Kutta

$$f'(0) = 0.969600$$

expressions for $\delta, \delta^*, \Theta, C_f, C_D, H$

Falkner-Skan wedge flows

$$U(x) = Kx^m$$

$$\eta = \gamma \sqrt{\frac{U(m+1)}{2rx}}, \quad \beta = \frac{2m}{m+1}$$

$$f''' + ff'' + \beta[1 - (f')^2] = 0$$

$$f(0) = 0, f'(0) = 0, f'(n \rightarrow \text{large}) = 1$$

s.t. $f''(0), f(n)$ for each β

$$U = U(x) f'(n), \text{ expressions for } \delta, \delta^*, \Theta, C_f, H$$

$m < 0 \Rightarrow$ decelerating freestream, thick,
low T_w , relatively unstable,
possible separation

$m > 0 \Rightarrow$ accelerating freestream, thin,
high T_w , relatively stable

flat plate with suction / blowing

$$\text{similarity solution} \quad U_w^* = \frac{U_w}{U} \sqrt{Re_x}$$

$$U_w \sim \frac{1}{\sqrt{x}}$$

$U_w^* > 0 \Rightarrow$ injection / blowing, thick, low T_w ,
relatively unstable, possible separation

$U_w^* < 0 \Rightarrow$ suction, thin, high T_w , relatively
stable

nonsimilar solutions

linearly retarded flow of Howarth

$$U(x) = U_0 (1 - x/L)$$

uniform suction

approximate integral methods

for general $U(x)$

$$\frac{C_S}{2} = \frac{1}{U^2} \frac{\partial}{\partial x} (U \delta^*) + \frac{\partial \theta}{\partial x} + (2\theta + \delta^*) \frac{1}{U} \frac{\partial U}{\partial x} - \frac{v_w}{U}$$

for steady, $v_w = 0$

$$\frac{C_S}{2} = \frac{d\theta}{dx} + (2 + H) \frac{\partial}{\partial x} \frac{dU}{dx}$$

method of Thwaites

$$\lambda = \frac{\partial^2}{x^2} \frac{dU}{dx} \quad \text{correlation parameter}$$

$$\frac{v_w \partial}{U U} = S(\lambda) \quad \text{shear correlation}$$

$$H = H(\lambda) \quad \text{shape factor correlation}$$

$$\lambda(x) = \frac{0.45 (\partial U / \partial x)}{U^6} \int_0^x U^5 dx'$$

separation at $\lambda = -0.09$

numerical methods

Finite difference explicit/implicit
marching schemes

MECE 6373
VISCOUS FLOW II
Review for EXAM #1

Chap 5 The Stability of Laminar Flows

Sec 5-1 Introduction: The Concept of Small-Disturbance Stability

If slightly disturb a system, will it return to its original state?
⇒ yes, then stable
⇒ no, then unstable

typical stability analysis

- establish controlling equations, formulate basic solution, Q_0
- add a disturbance to basic solution and substitute into the controlling equations
$$Q = Q_0 + Q_1, Q_1 \ll Q_0$$
- subtract out basic solution
- linearize ($Q_1^2 \ll Q_1$)
- solve resultant linear, homogeneous equations and assess growth, maintenance, or decay of Q_1 with time or space

example: one dim, compressible, stationary fluid
(supplementary)

example: buckling of simple beam under compression
(text)

example: forced motion of a damped mass with a cubic spring (text)

Sec 5-2 Linearized Stability of Parallel Viscous Flows

Navier-Stokes equations with ρ, μ constant

mass

$$\nabla \cdot \vec{V} = 0$$

mom

$$\frac{D\vec{V}}{Dt} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \vec{V} \quad \text{Eq (5-10)}$$

basic solution $\Rightarrow U, V, W, P_0$

disturbance $\Rightarrow \hat{U}, \hat{V}, \hat{W}, \hat{P}$

two-dim, parallel (boundary layer) flows

$$U \approx U(y)$$

$$V \ll U$$

$$W = 0$$

Squire's theorem: for two-dim, parallel flow

\Rightarrow minimum value of Re却it corresponds to

disturbances propagating in the flow direction

since linear, homogeneous equations with coefficients dependent only on y , expect traveling-wave solutions of the form

$$\hat{\xi}(x, y, t) = \xi(y) \exp[i\alpha(x - ct)]$$

Tollmein-Schlichting waves $\alpha = \text{wavenumber } (\alpha = 2\pi/\lambda)$

$c = \text{propagation speed}$

$\omega = \text{angular frequency } (\omega = \alpha c)$

$$i\alpha u + v' = 0$$

Eqs (5-15 a, b, c)

$$i\alpha u(U - c) + U'v = -\frac{i}{\rho} \alpha p + \nu (u'' - \alpha^2 u)$$

$$i\alpha v(U - c) = -\frac{1}{\rho} p' + \nu (v'' - \alpha^2 v)$$

$$(\xi' = \frac{d\xi}{dy})$$

combine Eqs (5-15a, b, c) \Rightarrow Orr-Sommerfeld equ.

$$(U - c)(v'' - \alpha^2 v) - U'' v + \frac{i\nu}{\alpha} (v''' - 2\alpha^2 v'' + \alpha^4 v) = 0$$

Eq (5-18)

if inviscid ($\nu=0, Re \Rightarrow \infty$) flow then

$$v'' - \left(\frac{U''}{U - c} + \alpha^2 \right) v = 0 \quad Eq (5-21)$$

for instability necessary that velocity profile have a point of inflection (PI), $|U'|_{PI}$ be a maximum, and $U''(U - U_{PI}) < 0$ somewhere on the profile

if viscous ($\nu \neq 0, Re \Rightarrow \text{finite}$), nondim Orr-Sommerfeld equ.

$$(U^* - c^*)(\phi'' - \alpha_s^2 \phi) - U^{*''} \phi + \frac{i}{\alpha_s Re} (\phi''' - 2\alpha_s^2 \phi'' + \alpha^4 \phi) = 0$$

Eq (5-23)

for given $U^*(x, y)$ and $Re_s \Rightarrow$ eigenvalue problem
eigenvalue equation

$$f(Re_s, \alpha, c) = 0$$

temporal stability $\alpha = 2\pi/\lambda \Rightarrow$ real and "known"

$C = Cr + iCs \Rightarrow$ complex and to be determined

if $C_s > 0 \Rightarrow$ growth, unstable

$C_s < 0 \Rightarrow$ decay, stable

$C_s = 0 \Rightarrow$ neutral

spatial stability

$\omega = \alpha c \Rightarrow$ real and "known"

$\alpha = \alpha_r + i\alpha_i \Rightarrow$ complex and
to be determined

if $\alpha_i < 0 \Rightarrow$ growth, unstable

$\alpha_i > 0 \Rightarrow$ decay, stable

$\alpha_i = 0$ neutral

stability analysis for Blasius ($U_\infty = \text{const.}$) and
Falkner-Skan ($U_\infty = Kx^m$) flows

similarity solutions, $U(x, y) \Rightarrow$ known

neutral stability curves \Rightarrow Figs. 5-6 and 5-7

x_{crit} \Rightarrow location where T-S waves first propagate
with increasing amplitude ($C_s > 0, \alpha_s < 0$),
first indication of instability

for flat plate, $R_{x, \text{crit}} \approx 91,000$

accelerating flows \Rightarrow stabilizing
($dU/dx > 0, dP/dx < 0$)

deaccelerating flows \Rightarrow destabilizing
($dU/dx < 0, dP/dx > 0$)

comparison of stability theory with experiment

Sec 5-3 Parametric Effects in the Linear
Stability Theory

classical laminar flow profiles (internal flows)

suction \Rightarrow stabilizing

shape factor correlation of Wazan (1979)

$R_{x, \text{crit}}^* = \text{func}(H) \Rightarrow$ Fig 5-12

to find $x_{crit} \rightarrow$ if Falkner-Skan \Rightarrow Table 5-1
otherwise Fig 5-12 and method of Thwaites
waves, jets, parallel fluids \Rightarrow very unstable
(Pi is profiles)
wall temperature \rightarrow gases, cold wall stabilizing
 \rightarrow liquids, hot wall stabilizing
compressible effects ($\rho = \rho_0 + \rho_1$, $T = T_0 + T_1$, etc.)
 \Rightarrow stabilizing
 \Rightarrow higher modes
 \Rightarrow nonparallel wave growth

compliant boundaries \Rightarrow can be stabilizing

free convection flows \Rightarrow relatively unstable
(Pi is profile)

centrifugal stability \Rightarrow concentric rotating cylinders,
cylindrical coordinate anal.,
Taylor number, Taylor
vortices

Sec 5-4 Transition to Turbulence

transition from first amplification of T-S waves
to full turbulence not well understood theoretically
rely great deal on visualization

flow over a flat plate, sequence of events

- at x_{crit} , small natural disturbances excite amplifying two-dim. T-S waves
- when T-S waves grow to about 1% of background properties, three dimensional (spanwise) waves occur (Λ vorties)
- Λ vorties begin to deteriorate

- sporadic, finite regions of full turbulence spontaneously occur \Rightarrow "turbulent spots"
- coalescence of turbulent spots into fully turbulent flow

Sec 5-5 Engineering Prediction of Transition

x_{tr} = location of transition to full turbulence
empirical correlations

two-step method of Granville (1953)
one-step method of Michel (1952)

Orr-Sommerfeld method of Tafte

transition when T-S waves amplify

$$\frac{A}{A_0} \approx 10^4 \approx e^9$$

purely theoretical prediction

Cebeci and Smith correlation (1974)

one-step method of Wazan (1979, 1981)

$$Re_{x,tr} = f_{unc}(H)$$

author prefers method of Wazan

effect of freestream turbulence

\Rightarrow hastens onset of turbulence

turbulence level $T = \frac{1}{U} \left[\frac{1}{3} (\bar{u}'^2 + \bar{v}'^2 + \bar{w}'^2) \right]^{1/2}$

for Falkner-Skan \Rightarrow van Driest and Blumer (1963)
correlation

otherwise \Rightarrow correlation of Durham with
method of Thwaites

effect of surface roughness \Rightarrow can hasten transition
"trip" wire criterion

$$U_t/k_r \geq 850$$

$$k/s^* \geq 0.3$$

oscillating boundary layers $U_\infty = U_0 (1 + N_A \sin \omega t)$

unsteady Re number, $R_{ens} = \frac{N_A U_0^2}{\nu}$

hastens transition if $R_{ens} \geq 27,000$

accelerating flow in a pipe \Rightarrow very stabilizing
^{can be}

chaos \Rightarrow applications in turbulent modeling