

$$k_n L = n\pi \quad n = 1, 2, 3$$

$$\frac{2\pi f_n L}{c} = n\pi$$

$$f_n = \left(\frac{n}{2}\right) \left(\frac{c}{L}\right) \quad \text{Eq (3.4.10)}$$

∫ natural frequencies of a free-free bar  
(overtones are harmonics)

if  $m \gg m_b$  (fixed end)

$$\tan(k_n L) \Rightarrow -\infty$$

$$\cos(k_n L) = 0$$

$$k_n L = \frac{(2n-1)\pi}{2}$$

$$\frac{2\pi f_n L}{c} = \frac{(2n-1)\pi}{2}$$

$$f_n = \frac{(2n-1)}{4} \left(\frac{c}{L}\right) \quad \text{Eq (3.4.13)}$$

∫ nat. freq. of a free-fixed bar  
(overtones are harmonics)  
(note: no even harmonics of fund.)

otherwise

$$\tan(k_n L) = -\left(\frac{m}{m_b}\right) (k_n L)$$

$f_n$  not<sup>nec.</sup> harmonics of fundamental!  
(overtones not harmonic)

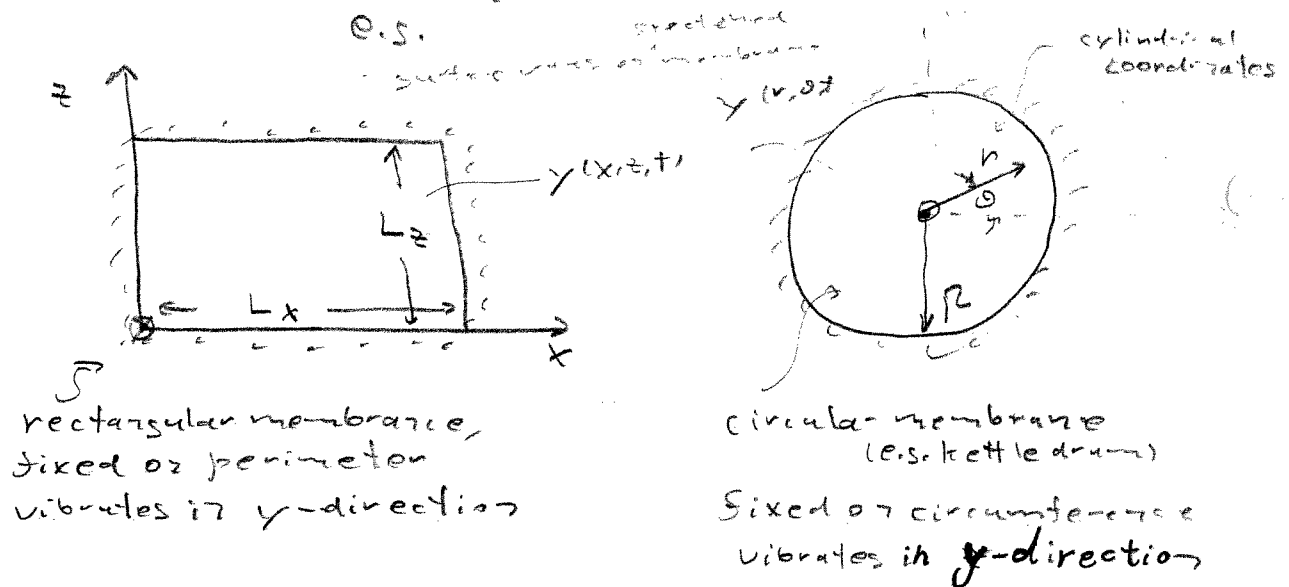
finish Chap 3

advanced topics → flexural waves  
→ shear waves

## Chap 4 The Two-Dimensional Wave Equation: Vibrations of Membranes

So far have considered one-dimensional waves (waves that can propagate only in one spatial direction) and the corresponding one-dimensional form of the wave equation

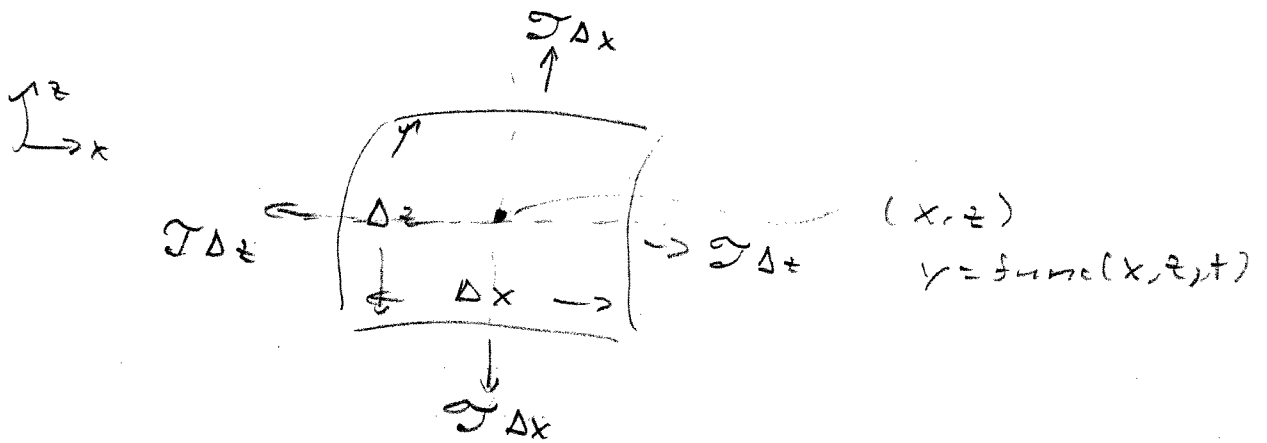
as we gradually progress towards the full three dimensional wave equation, in Chap 4 consider transverse vibrations of a stretched membrane  
→ two-dim wave equation



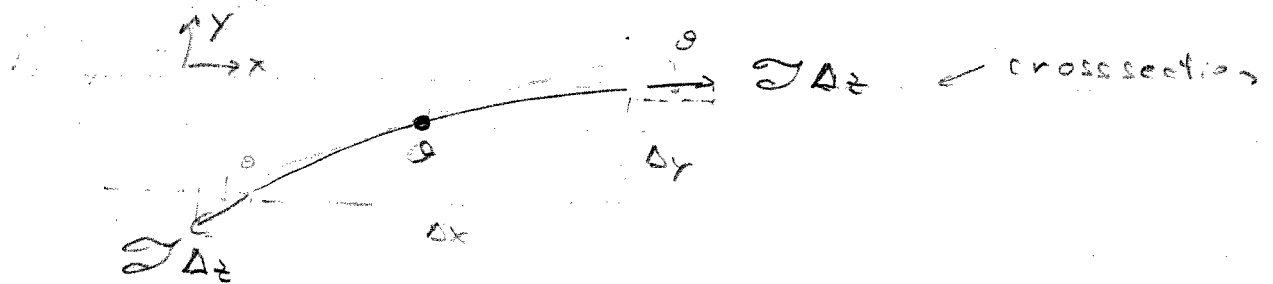
assume ⇒ (as with strings)  
 small displacements  
 uniform surface tension ( $\gamma$ )  
 uniform surface density ( $\rho_s = \text{mass/area}$ )  
 neglect gravitational effects

consider free body diagram of incremental surface element of membrane

(rectangular coordinates)



$\sigma$  = surface tension of membrane  $\left[ \frac{\text{force}}{\text{length}} \right], \left[ \frac{\text{N}}{\text{m}} \right]$



$$\begin{aligned} \Delta F_y)_x &= (\sigma \sin \theta \Delta z)_{x+\Delta x/2} - (\sigma \sin \theta \Delta z)_{x-\Delta x/2} \\ &= \frac{\partial (\sigma \sin \theta)}{\partial x} (\Delta x \Delta z) = \frac{\partial}{\partial x} \left( \sigma \frac{\partial y}{\partial x} \right) \Delta x \Delta z \\ &= \sigma \frac{\partial^2 y}{\partial x^2} \Delta x \Delta z \end{aligned}$$

likewise

$$\Delta F_y)_z = \sigma \frac{\partial^2 y}{\partial z^2} \Delta x \Delta z$$

Newton's 2<sup>nd</sup> law of motion

$$\begin{aligned} (\rho_s \Delta x \Delta z) \frac{\partial^2 y}{\partial t^2} &= \Delta F_y \\ &= \sigma \left[ \frac{\partial^2 y}{\partial x^2} + \frac{\partial^2 y}{\partial z^2} \right] \Delta x \Delta z \end{aligned}$$

$$\frac{\partial^2 y}{\partial t^2} = \left( \frac{\sigma}{\rho_s} \right) \left( \frac{\partial^2 y}{\partial x^2} + \frac{\partial^2 y}{\partial z^2} \right)$$

define  $c = \sqrt{\frac{g}{\rho_s}}$  = phase speed (CT as  $\mathcal{D}$ ,  $c = \dots$ )

so

$$\frac{\partial^2 y}{\partial t^2} = c^2 \left( \frac{\partial^2 y}{\partial x^2} + \frac{\partial^2 y}{\partial z^2} \right) \quad \text{Eq (A.2.2)}$$

$\mathcal{P}$  two-dimensional wave equation

or can introduce a vector calculus operator

$$\nabla^2 = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial z^2} \quad (\text{two-dim}) \text{ Laplacian operator}$$

so equivalently

$$\nabla^2 y = \frac{1}{c^2} \frac{\partial^2 y}{\partial t^2} \quad \text{Eq (A.2.4)}$$

$\mathcal{P}$  even though derived in rectangular coordinates, in vector calculus form, valid for any orthogonal coordinate system, as long as use correct definition of  $\nabla^2$

e.g., if use cylindrical <sup>(polar)</sup> coordinates (see App A7)

$$\nabla^2 = \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \quad y(r, \theta, t)$$

(simple harmonic)

now for sinusoidal solutions, expect

$$\bar{y} = \bar{\psi} e^{j\omega t} \quad \text{so } \bar{\psi} = \text{func}(x, z) \text{ or func}(r, \theta)$$

$$\nabla^2 \bar{\psi} e^{j\omega t} = -\frac{\omega^2}{c^2} \bar{\psi} e^{j\omega t}$$

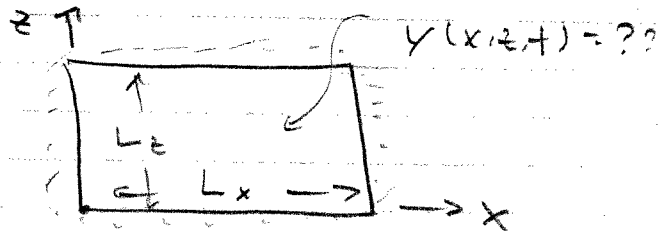
$$k = \omega/c$$

$$\nabla^2 \bar{\psi} + k^2 \bar{\psi} = 0 \quad \text{Eq (A.2.9)}$$



## Helmholtz equation (time-independent wave equation)

consider now unforced vibration of a rectangular membrane with a fixed rim



B.C.'s  $y(0, z, t) = 0$   $y(L_x, z, t) = 0$

$y(x, 0, t) = 0$   $y(x, L_z, t) = 0$

I.C.'s  $y(x, z, 0)$  unknown  $\partial y / \partial t(x, z, 0)$  unknown  
assume solution of the form

$$\bar{y}(x, z, t) = \bar{\Psi} e^{j\omega t}$$

(can build up more general solutions by summing)

then 
$$\frac{\partial^2 \bar{\Psi}}{\partial x^2} + \frac{\partial^2 \bar{\Psi}}{\partial z^2} + k^2 \bar{\Psi} = 0$$

try to solve by separation of variables, assume a solution of the form

$$\bar{\Psi}(x, z) = \bar{X}(x) \bar{Z}(z) \text{ then}$$

$$\bar{Z} \frac{d^2 \bar{X}}{dx^2} + \bar{X} \frac{d^2 \bar{Z}}{dz^2} + k^2 \bar{X} \bar{Z} = 0$$

or

$$\frac{1}{\bar{Z}} \frac{d^2 \bar{Z}}{dz^2} = -k^2 - \frac{1}{\bar{X}} \frac{d^2 \bar{X}}{dx^2} = \text{constant} = -k_z^2$$

$\int$   
func(z)  
only

$\int$   
func(x)  
only

so 
$$\frac{d^2 \bar{Z}}{dz^2} + k_z^2 \bar{Z} = 0 \quad \text{also}$$

$$\frac{1}{\bar{X}} \frac{d^2 \bar{X}}{dx^2} = -k_x^2 + k_z^2 = -k_x^2$$

where  $k_x^2 + k_z^2 = k^2$  Eq (4.3.7)

so  $\frac{d^2 \bar{X}}{dx^2} + k_x^2 \bar{X} = 0$

general solutions

$$\bar{X}(x) = \bar{C}_1 \cos(k_x x) + \bar{C}_2 \sin(k_x x)$$

$$\bar{Z}(z) = \bar{C}_3 \cos(k_z z) + \bar{C}_4 \sin(k_z z)$$

B.C.'s  $\bar{Y}(0, z, t) = \bar{C}_1 \bar{Z}(z) e^{j\omega t} = 0 \rightarrow \bar{Y}(x, z, t) = \bar{X}(x) \bar{Z}(z) e^{j\omega t}$

$$\bar{C}_1 = 0$$

$$\bar{Y}(x, 0, t) = \bar{C}_3 \bar{X}(x) e^{j\omega t} = 0$$

$$\bar{C}_3 = 0$$

$$\bar{Y}(x, z, t) = \bar{A} \sin(k_x x) \sin(k_z z) e^{j\omega t}$$

$$\bar{Y}(L_x, z, t) = \bar{A} \sin(k_x L_x) \sin(k_z z) e^{j\omega t} = 0$$

$$k_x L_x = n\pi \quad n=1, 2, 3, \dots$$

$$k_{x,n} = \frac{n\pi}{L_x} \quad \text{eigenvalue}$$

$$\bar{Y}(x, L_z, t) = \bar{A} \sin(k_x x) \sin(k_z L_z) e^{j\omega t} = 0$$

$$k_{z,m} = \frac{m\pi}{L_z} \quad m=1, 2, 3, \dots$$

thus

$$\bar{Y}_{n,m}(x, z, t) = \bar{A}_{n,m} \sin\left(n\pi \frac{x}{L_x}\right) \sin\left(m\pi \frac{z}{L_z}\right) e^{j\omega_{nm} t}$$

eigenfunction / natural mode / normal mode

where  $k_{n,m}^2 = k_{x,n}^2 + k_{z,m}^2$

$$= \left(\frac{n\pi}{L_x}\right)^2 + \left(\frac{m\pi}{L_z}\right)^2 = \frac{\omega_{n,m}^2}{c^2}$$

$$\omega_{n,m} = c \left[ \left(\frac{n\pi}{L_x}\right)^2 + \left(\frac{m\pi}{L_z}\right)^2 \right]^{1/2} = 2\pi f_{n,m}$$

$$f_{n,m} = \left(\frac{c}{2}\right) \left[ \left(\frac{n}{L_x}\right)^2 + \left(\frac{m}{L_z}\right)^2 \right]^{1/2}$$

eigenfrequencies (natural / normal frequencies)

$$m = 1, 2, 3, \dots$$

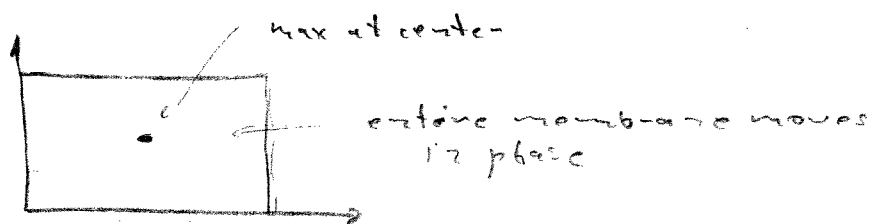
$$n = 1, 2, 3, \dots$$

Fundamental (lowest)  $f_{1,1} = \frac{c}{2} \left[ \left(\frac{1}{L_x}\right)^2 + \left(\frac{1}{L_z}\right)^2 \right]^{1/2}$

overtones not nec. harmonic!

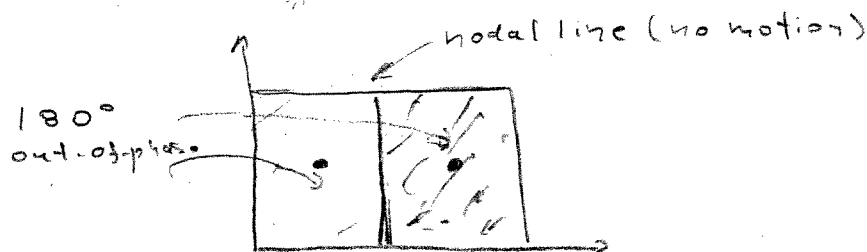
(1,1) mode

$$\bar{y}_{1,1} \sim \sin\left(\pi \frac{x}{L_x}\right) \sin\left(\pi \frac{z}{L_z}\right)$$



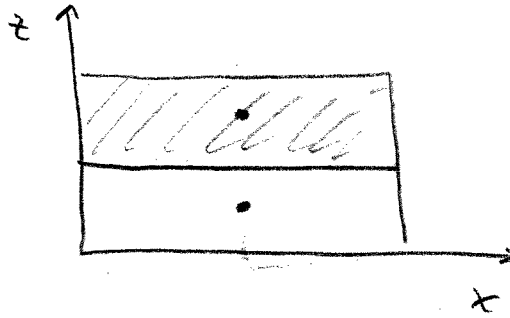
(2,1) mode

$$\bar{y}_{2,1} \sim \sin\left(2\pi \frac{x}{L_x}\right) \sin\left(\pi \frac{z}{L_z}\right)$$

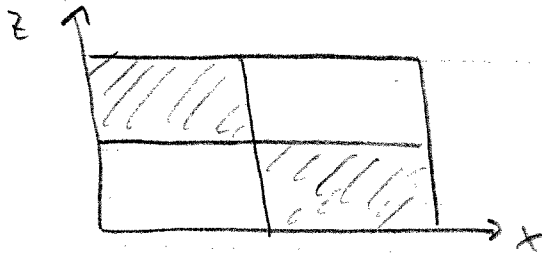




(1,2)



(2,2)



modes are orthogonal

etc.

(I.C.)

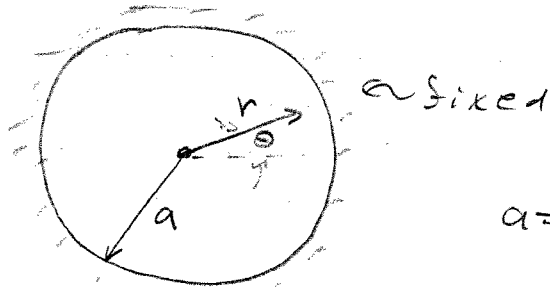
$$\bar{y} = \sum_{m,n} \bar{Y}_{mn}$$

choose  $\bar{A}_{mn}$  to sat. initial con.

any general unforced motion can be expressed by summing over these natural modes

now consider general vibration of a circular membrane (e.g. a kettle drum) with fixed rim

cylindrical coordinates



a = radius

as before

$$\nabla^2 y = \frac{1}{c^2} \frac{\partial^2 y}{\partial t^2}$$

but now  $y = y(r, \theta, t)$

if assume

$$\bar{y}(r, \theta, t) = \bar{\Psi}(r, \theta) e^{i\omega t} \text{ then}$$

$$\nabla^2 \bar{\Psi} + k^2 \bar{\Psi} = 0$$

Lecture #15 Vibrating Membrane /  
Review for EXAM #1

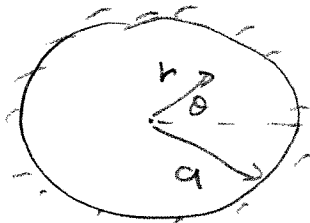
Read Secs. 5.1 → 5.5 (for Friday)

EXAM #1, Wed., Feb. 16, Chap 1, 2, 3, 4

— return H.W. #9, collect H.W. #10

— will discuss/review for exam in second half of period, first finish up Chap 4

last time ⇒ circular membrane, fixed rim



cyl. coord

$$\Psi(r, \theta, t) = \bar{\Psi}(r, \theta) e^{j\omega t}$$

$$\nabla^2 \bar{\Psi} + k^2 \bar{\Psi} = 0$$

$$\frac{\partial^2 \bar{\Psi}}{\partial r^2} + \frac{1}{r} \frac{\partial \bar{\Psi}}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \bar{\Psi}}{\partial \theta^2} + k^2 \bar{\Psi} = 0$$

in cylindrical coordinates

$$\overset{\text{two-dim}}{\nabla^2} = \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} \quad \text{so}$$

$$\frac{\partial^2 \bar{\Psi}}{\partial r^2} + \frac{1}{r} \frac{\partial \bar{\Psi}}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \bar{\Psi}}{\partial \theta^2} + k^2 \bar{\Psi} = 0$$

try again: separation of variables solution where

assume  $\bar{\Psi}(r, \theta) = \bar{R}(r) \bar{\Theta}(\theta)$

$$\frac{\bar{\Theta}}{\bar{R}\bar{\Theta}} \frac{d^2 \bar{R}}{dr^2} + \frac{\bar{\Theta}}{\bar{R}\bar{\Theta}} \frac{1}{r} \frac{d\bar{R}}{dr} + \frac{\bar{R}}{\bar{R}\bar{\Theta}} \frac{1}{r^2} \frac{d^2 \bar{\Theta}}{d\theta^2} + k^2 \bar{R}\bar{\Theta} = 0$$

$$r^2 \left\{ \frac{1}{\bar{R}} \frac{d^2 \bar{R}}{dr^2} + \frac{1}{\bar{R}} \frac{1}{r} \frac{d\bar{R}}{dr} + k^2 \right\} = - \frac{1}{\bar{\Theta}} \frac{d^2 \bar{\Theta}}{d\theta^2} = \text{constant} = +m$$

thus  $\frac{d^2 \bar{\Theta}}{d\theta^2} + m^2 \bar{\Theta} = 0$

$$\bar{\Theta}(\theta) = \bar{C}_1 \cos(m\theta + \gamma)$$

but by symmetry  $\bar{\Theta}(\theta \pm 2\pi) = \bar{\Theta}(\theta)$

thus  $m = \text{integer} = 0, 1, 2, \dots$

then

$$\frac{d^2 \bar{R}}{dr^2} + \frac{1}{r} \frac{d\bar{R}}{dr} + \left( k^2 - \frac{m^2}{r^2} \right) \bar{R} = 0 \quad \text{Eq (4.4.8)}$$

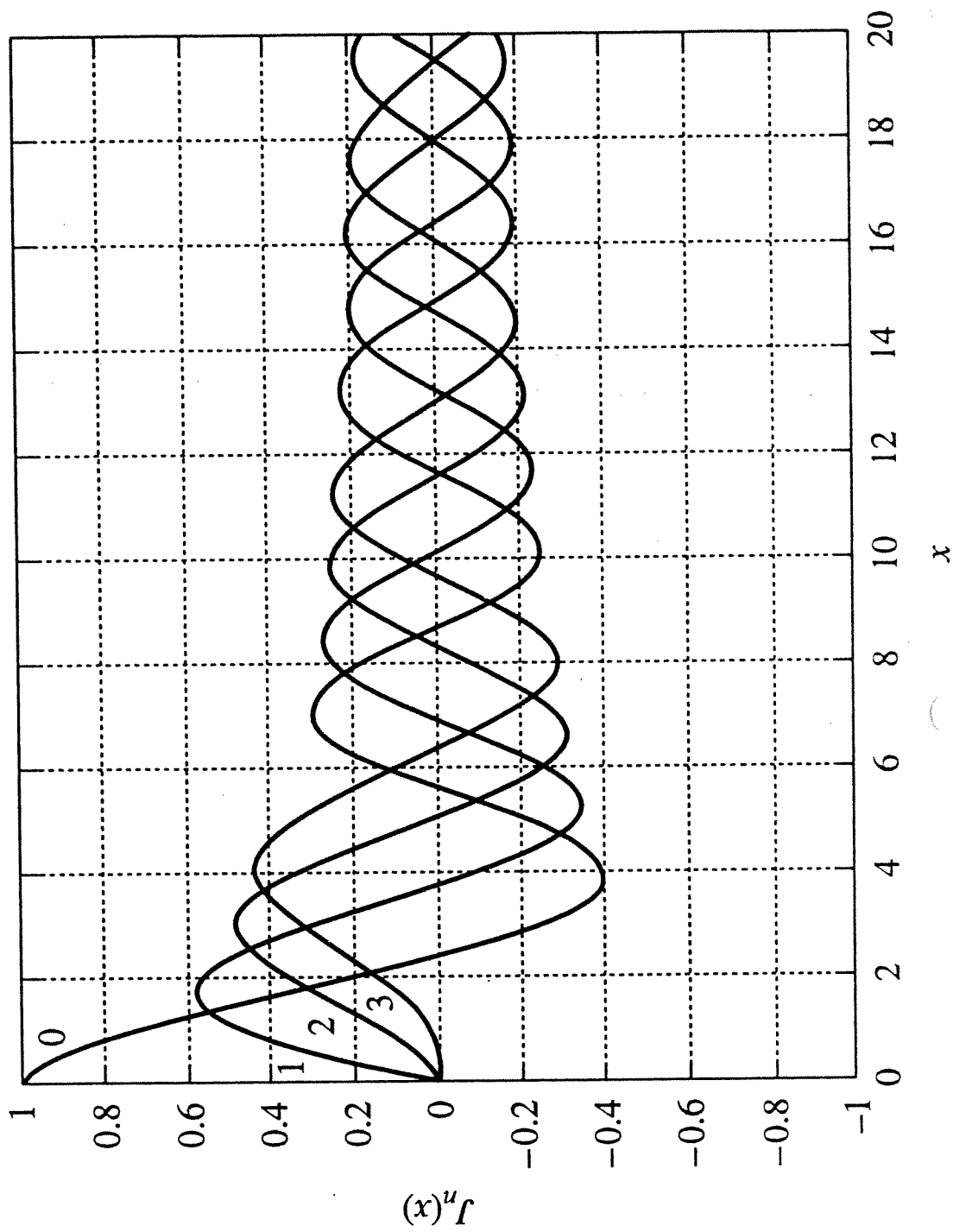
$m^{\text{th}}$ -order  
well-known ord. diff. eqn., Bessel's eqn., *important solution*

see App A4 and A5

general solution

$$\bar{R}(r) = \bar{C}_2 J_m(kr) + \bar{C}_3 Y_m(kr)$$

*(b) Graphs: Bessel Functions of the First Kind of Orders 0, 1, 2, and 3*



(c) Zeros: Bessel Functions of the First Kind,  $J_m(j_{mn}) = 0$

		$j_{mn}$				
$n \backslash m$	0	1	2	3	4	5
0	—	2.40	5.52	8.65	11.79	14.93
1	0	3.83	7.02	10.17	13.32	16.47
2	0	5.14	8.42	11.62	14.80	17.96
3	0	6.38	9.76	13.02	16.22	19.41
4	0	7.59	11.06	14.37	17.62	20.83
5	0	8.77	12.34	15.70	18.98	22.22

$J_m(kr) \Rightarrow m^{\text{th}}$  order Bessel function of 1<sup>st</sup> kind

B.C.s  $Y_m(kr) \Rightarrow m^{\text{th}}$  order Bessel function of 2<sup>nd</sup> kind

note  $Y_m(kr) \Rightarrow \pm\infty$  as  $kr \Rightarrow 0$

for bounded solutions, require that  $\bar{C}_3 = 0$

so

$$\bar{y}(r, \theta, t) = \bar{A} J_m(kr) \cos(m\theta + \gamma) e^{j\omega t}$$

$$\text{@ } r=a \quad \bar{y}(a, \theta, t) = \bar{A} J_m(ka) \cos(m\theta + \gamma) e^{j\omega t} = 0$$

$$\text{thus } J_m(ka) = 0$$

$$k_{min} a = j_{m,1} \Rightarrow \underline{k_{min}} = \frac{j_{m,1}}{a} = \frac{\omega_{min}}{c} = \frac{2\pi f}{c}$$

where  $J_m(j_{m,n}) = 0$

$j_{m,n}$  = zeroes of  $J_m$

such that  $j_{m,1} < j_{m,2} < j_{m,3} < j_{m,4} < \dots$

see Table A5 c.) page 516

$$\text{thus } \omega_{min} = \left(\frac{c}{a}\right) j_{m,1}$$

$$f_{min} = \frac{1}{2\pi} \left(\frac{c}{a}\right) j_{m,1}$$

$$\text{fundamental frequency } f_{0,1} = \frac{1}{2\pi} \left(\frac{c}{a}\right) j_{0,1}$$

$$= \left(\frac{1}{2\pi}\right) \left(\frac{c}{a}\right) (2.40)$$

in general overtones not harmonic!  $= 0.7639 \left(\frac{c}{2a}\right)$

$$\text{e.g. } \frac{j_{1,1}}{j_{0,1}} = \frac{3.83}{2.40} \neq \text{integer, etc.}$$

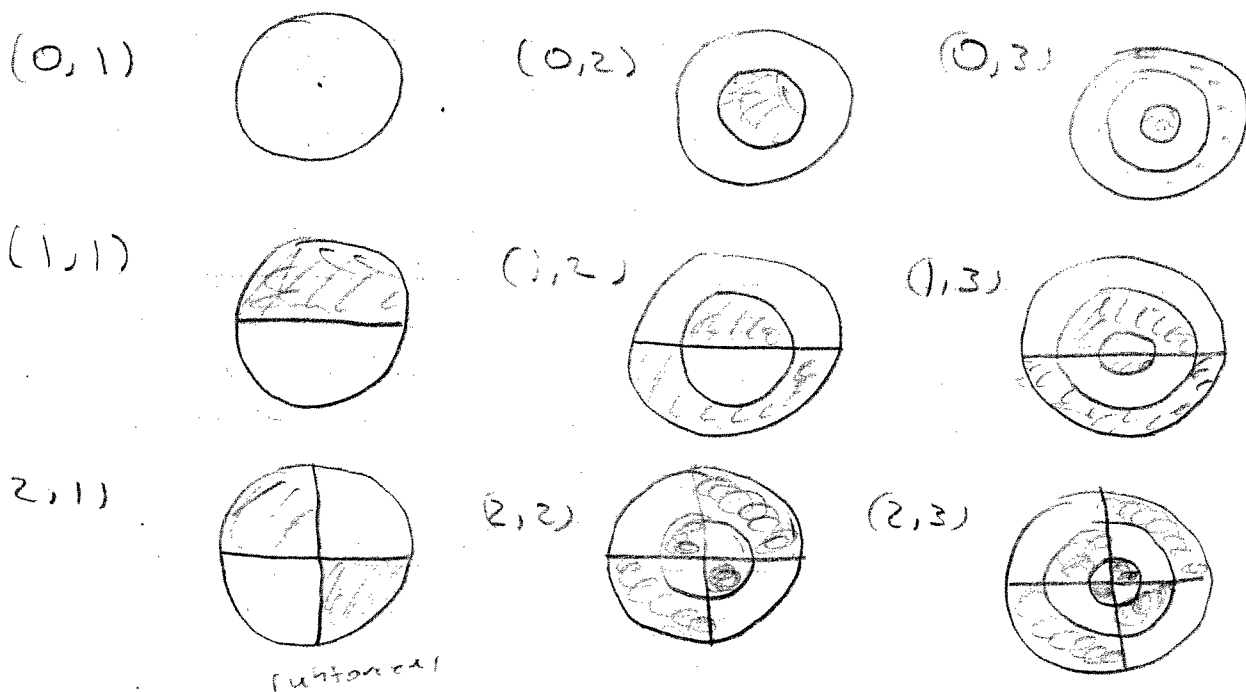
so natural mode

$$\bar{Y}_{mn}(r, \theta, t) = \bar{A}_{mn} \bar{J}_m(k_{mn}r) \cos(m\theta + \gamma_{mn}) e^{j\omega_{mn}t}$$

$m$  = number of radial nodal lines  
(number of  $\theta$  cycles)

$n$  = number of circular nodal lines  
(including rim)

see Fig 4.4.1



any general motion can be written as a sum of these natural (normal) modes

(non-symmetric modes not considered)

finish up by discussing purely radial modes

 $m=0$  then 

$$\bar{Y}_{0,n}(r, \theta, t) = \bar{A}_{0,n} J_0(k_{0,n} r) \cos(\gamma_{0,n}) e^{j\omega_n t}$$

or  $\bar{A}_n = A_n e^{j\phi_n}$ 

$$Y_n(r, t) = A_n J_0(k_n r) \cos(\omega_n t + \phi_n)$$

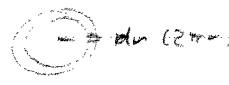
$$k_n = \frac{j_{0,n}}{a} = \frac{\omega_n}{c}$$

general solution

$$Y(r, t) = \sum_{n=1}^{\infty} A_n J_0(k_n r) \cos(\omega_n t + \phi_n)$$

since  $J_0(0) = 1$ ,  $A_n \Rightarrow$  amplitude of  $n^{\text{th}}$  mode at  $r=0$ average displacement amplitude for  $n^{\text{th}}$  mode

$$\langle \Psi_n \rangle_S = \frac{1}{\pi a^2} \int_0^a A_n J_0(k_n r) 2\pi r dr$$

$\leftarrow dA$  

now  $\alpha J_0(\alpha) = \frac{d[\alpha J_1(\alpha)]}{d\alpha}$  App A4

$$\alpha = k_n r \Rightarrow r = \frac{\alpha}{k_n} \quad dr = \frac{d\alpha}{k_n}$$

$$\begin{aligned} \langle \Psi_n \rangle_S &= \frac{1}{\pi a^2} \int_0^{k_n a} A_n J_0(\alpha) 2\pi \frac{\alpha}{k_n} \frac{d\alpha}{k_n} \\ &= \frac{1}{\pi a^2} 2\pi A_n \frac{1}{k_n^2} \int_0^{k_n a} \alpha J_0(\alpha) d\alpha \\ &= \frac{1}{\pi a^2} 2\pi A_n \frac{1}{k_n^2} (k_n a) J_1(k_n a) \end{aligned}$$



$$\begin{aligned} \langle \psi_n \rangle_s &= \frac{2A_n}{(kna)} J_1(kna) \quad E_2 \quad (4.5.4) \\ &= \frac{2A_n}{j_{0,n}} J_1(j_{0,n}) \end{aligned}$$

$$\begin{aligned} \text{Sind } \langle \psi_1 \rangle_s &= \frac{2A_1}{2.40} J_1(2.40) = \frac{2(0.5202)}{2.40} A_1 \\ &= 0.434 A_1 \end{aligned}$$

$$\begin{aligned} \langle \psi_2 \rangle_s &= \frac{2A_2}{5.52} J_1(5.52) \quad \left\{ \begin{array}{l} -0.3400 \\ 0.2712 \end{array} \right. \\ &= -0.123 A_2 \end{aligned}$$

$$\begin{aligned} \langle \psi_3 \rangle_s &= \frac{2A_3}{8.65} J_1(8.65) \quad \left\{ \begin{array}{l} 0.2712 \\ -0.3400 \end{array} \right. \\ &= 0.0627 A_3 \\ &\quad \text{etc.} \end{aligned}$$

higher order modes displace smaller volume of surrounding fluid, less acoustic emission

any last questions about Chap 4, transverse vibrations of stretched membrane?

rest of Chap 4  $\Rightarrow$  vibrations  
surface waves in finite plate  
etc.

Intro. to

Lecture #17 Acoustics of Fluids

Read Secs. 5.6 → 5.7

- return homework

- have exams graded, class as a whole did o.k.  
will return and discuss near end of period

- now ⇒ Chap 5 Intro. to Acoustics of Fluids